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Pole Placement and Active Vibration Control in Aeroelasticity

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Receptance Method

• Dynamic stiffnesses → Receptances:

- No need to evaluate or to know the system matrices M, C, K.
- Any input-output transfer function may be used dynamics of actuators, sensors, filters etc. included in the measurement.

Receptance Method Partial Pole Placement Problem

Open-loop and closed-loop systems:

$$\lambda_k^2 \mathbf{M} + \lambda_k \mathbf{C} + \mathbf{K} \mathbf{v}_k = \mathbf{0}$$

$$\mu_k^2 \mathbf{M} + \mu_k \mathbf{C} + \mathbf{K} \mathbf{w}_k = \mathbf{B} \mathbf{u}(t)$$

$$\mathbf{u}(t) = \mu_k \mathbf{F}^T + \mathbf{G}^T \mathbf{w}_k$$

Assigned eigenvalues $\mu_k \stackrel{p}{}_{k=1}^{p}$ are distinct from eigenvalues $\lambda_k \stackrel{2n}{}_{k=1}^{k}$ While eigenvalues $\mu_k = \lambda_k \quad k = p+1, p+2, \dots 2n$ are unchanged.

Unchanged Eigenvalues

$$\lambda_k^2 \mathbf{M} + \lambda_k \mathbf{C} + \mathbf{K} \ \mathbf{w}_k = \mathbf{B} \ \lambda_k \mathbf{F}^T + \mathbf{G}^T \ \mathbf{w}_k$$
$$\mathbf{B} = \begin{bmatrix} \mathbf{b}_1 & \mathbf{b}_2 & \cdots & \mathbf{b}_m \end{bmatrix} \quad \mathbf{F} = \begin{bmatrix} \mathbf{f}_1 & \mathbf{f}_2 & \cdots & \mathbf{f}_m \end{bmatrix} \quad \mathbf{G} = \begin{bmatrix} \mathbf{g}_1 & \mathbf{g}_2 & \cdots & \mathbf{g}_m \end{bmatrix}$$

May be re-written as,

$$\mathbf{\mathbf{\mu}}_{k}^{2}\mathbf{M} + \mathbf{\mathbf{\mu}}_{k}\mathbf{C} + \mathbf{K}\mathbf{\mathbf{\mathbf{y}}}_{k} = \mathbf{\mathbf{\mathbf{\phi}}}_{1}\mathbf{\mathbf{\mathbf{\mu}}}_{k}\mathbf{f}_{1}^{T} + \mathbf{g}_{1}^{T} + \mathbf{b}_{2}\mathbf{\mathbf{\mathbf{\mu}}}_{k}\mathbf{f}_{2}^{T} + \mathbf{g}_{2}^{T} + \dots + \mathbf{b}_{m}\mathbf{\mathbf{\mathbf{\mu}}}_{k}\mathbf{f}_{m}^{T} + \mathbf{g}_{m}^{T}\mathbf{\mathbf{\mathbf{\mathbf{y}}}}_{k}\mathbf{\mathbf{\mathbf{y}}}_{k}$$

Non-trivial solution: $\mathbf{w}_k = \mathbf{v}_k$

$$\mathbf{\Phi}_1 \mathbf{\Phi}_k \mathbf{f}_1^T + \mathbf{g}_1^T + \dots + \mathbf{b}_m \mathbf{\Phi}_k \mathbf{f}_m^T + \mathbf{g}_m^T \mathbf{y}_k = \mathbf{0} \qquad k = p + 1, p + 2, \dots 2n$$

$$\begin{bmatrix} \lambda_k \mathbf{v}_k^T & \mathbf{0} & \cdots & \mathbf{0} & \mathbf{v}_k^T & \mathbf{0} & \cdots & \mathbf{0} \\ \mathbf{0} & \lambda_k \mathbf{v}_k^T & \cdots & \mathbf{0} & \mathbf{0} & \mathbf{v}_k^T & \cdots & \mathbf{0} \\ \vdots & \vdots \\ \mathbf{0} & \mathbf{0} & \cdots & \lambda_k \mathbf{v}_k^T & \mathbf{0} & \mathbf{0} & \cdots & \mathbf{v}_k^T \end{bmatrix} \begin{bmatrix} \mathbf{f}_1 \\ \vdots \\ \mathbf{f}_m \\ \mathbf{g}_1 \\ \vdots \\ \mathbf{g}_m \end{bmatrix} = \begin{pmatrix} \mathbf{0} \\ \mathbf{0} \\ \vdots \\ \mathbf{0} \end{pmatrix}$$

Assigned Eigenvalues

$$\mathbf{w}_{k} = \mathbf{H} \mathbf{v}_{k} \mathbf{p}_{1} \mathbf{v}_{k} \mathbf{f}_{1}^{T} + \mathbf{g}_{1}^{T} \mathbf{p}_{1} \mathbf{p}_{2} \mathbf{v}_{k} \mathbf{f}_{2}^{T} + \mathbf{g}_{2}^{T} \mathbf{p}_{2} \mathbf{p}_{1} \mathbf{p}_{2} \mathbf{p}_{k} \mathbf{p}_{2}^{T} \mathbf{p}_{2} \mathbf{p}_{$$

Let
$$\mathbf{r}_{\mu_k,j} = \mathbf{H} \mathbf{q}_{\mu_k} \mathbf{b}_j$$
 and $\alpha_{\mu_k,j} = \mathbf{q}_{\mu_k} \mathbf{f}_j^T + \mathbf{g}_j^T \mathbf{y}_k$ $j = 1, 2, ..., m$



where $\mathbf{W}_{k} = \alpha_{\mu_{k},1} \mathbf{r}_{\mu_{k},1} + \alpha_{\mu_{k},2} \mathbf{r}_{\mu_{k},2} + \dots + \alpha_{\mu_{k},m} \mathbf{r}_{\mu_{k},m}$

The closed-loop mode shape is determined by the choice of $\alpha_{\mu_k,j}$

Receptance Method

The control gains are then given by the solution of,

Y.M. Ram and J.E. Mottershead, Multiple-input active vibration control by partial pole placement using the method of receptances, *Mechanical Systems and Signal Processing*, 40, 2013, 727-735.

General Procedure

- Measure the open loop input-output FRF over a desired frequency range.
- Fit MIMO rational fraction polynomials to the measure FRF and obtain the input-output transfer functions.
- Select force distribution vectors $\mathbf{b}_k(s)$.
- Apply the Receptance Method to obtain unknown gains, \mathbf{g}_k , \mathbf{f}_k .

$$\begin{bmatrix} \mathbf{P}_{1} \\ \vdots \\ \mathbf{P}_{p} \\ \mathbf{Q}_{p+1} \\ \vdots \\ \mathbf{Q}_{2n} \end{bmatrix} \begin{pmatrix} \mathbf{f}_{1} \\ \vdots \\ \mathbf{f}_{m} \\ \mathbf{g}_{1} \\ \vdots \\ \mathbf{g}_{m} \end{pmatrix} = \begin{pmatrix} \boldsymbol{\alpha}_{1} \\ \vdots \\ \boldsymbol{\alpha}_{p} \\ \mathbf{0} \\ \vdots \\ \mathbf{0} \end{pmatrix}$$

• Implementation of the controller using dSPACE in real time.

Flutter Suppression Wind-Tunnel Aerofoil Rig

Aerofoil

Torsion Bar



V-stack piezo actuator

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Curve Fitting



Open loop FRFs include not only the dynamics of the aerofoil system but also the power amplifier, the actuator, the sensors and the effects of A/D and D/A conversion, numerical differentiation (Simulink/dSPACE) of displacements and high- and low-pass Butterworth filters with cut-off frequencies of 1Hz and 35 Hz.

Frequency/Damping Trends – Root Locus

Separation of pitch and heave frequencies



Poles assigned: $\mu_{pitch} = -1.5 \pm 38i$, $\mu_{heave} = -0.7 \pm 23i$

at wind speed, 7m/s.

Flutter Margin

$$\begin{split} F &= \left[\left(\frac{\widetilde{\omega}_2^2 - \widetilde{\omega}_1^2}{2} \right) + \left(\frac{\sigma_2^2 - \sigma_1^2}{2} \right) \right]^2 + 4\sigma_1 \sigma_2 \left[\left(\frac{\widetilde{\omega}_2^2 + \widetilde{\omega}_1^2}{2} \right) + 2\left(\frac{\sigma_2^2 + \sigma_1^2}{2} \right)^2 \right] \\ &- \left[\left(\frac{\sigma_2 - \sigma_1}{\sigma_2 + \sigma_1} \right) \left(\frac{\widetilde{\omega}_2^2 - \widetilde{\omega}_1^2}{2} \right) + 2\left(\frac{\sigma_2 + \sigma_1}{2} \right)^2 \right]^2 \end{split}$$



$$\widetilde{\omega}_{j} = \omega_{j} \sqrt{\left(1 - \zeta_{j}^{2}\right)}$$
$$j = 1,2$$
$$\sigma_{j} = -\zeta_{j} \omega_{j}$$

Zimmerman N.H. and Weissenburger J.T. (1964)

Quadratic flutter speed prediction.

Predicted flutter speed increased from 17 m/s to 20 m/s.

Separation of pitch and heave frequencies. $\mathbf{g} = -28.2 \quad 29.5^{T} \quad \mathbf{f} = 0.0284 \quad 0.0232^{T}$

Vibration control – 2 DOF aeroelastic system

Controller OFF - oscillation



Controller ON - oscillation eliminated

Tensioned-Wire Plunge Nonlinearity Feedback Linearisation

 $\dot{\mathbf{x}} = \mathbf{f} \ \mathbf{x} + \mathbf{g} \boldsymbol{u} \qquad \boldsymbol{y} = \boldsymbol{x}_1 \qquad \boldsymbol{u} = \boldsymbol{\delta}$

$$\begin{aligned} x_{1}' &= x_{2} \quad x_{2}' = f_{2} \quad \mathbf{x} + g_{2}u \quad \text{Pitch} \\ x_{3}' &= x_{4} \quad x_{4}' = f_{4} \quad \mathbf{x} + g_{4}u \quad \text{Plunge} \\ x_{5}' &= x_{1} - \varepsilon_{1}x_{5} \quad x_{6}' = x_{1} - \varepsilon_{2}x_{6} \\ x_{7}' &= x_{3} - \varepsilon_{1}x_{7} \quad x_{8}' = x_{3} - \varepsilon_{2}x_{8} \\ x_{9}' &= u - \varepsilon_{1}x_{9} \quad x_{10}' = u - \varepsilon_{2}x_{10} \\ x_{11}' &= -\varepsilon_{3}x_{11} \quad x_{12}' = -\varepsilon_{4}x_{12} \end{aligned}$$
 Aerodynamic



States x_5 and x_6 are associated with pitch, x_7 and x_8 are associated with plunge, x_9 and x_{10} are associated with flap motion and x_{11} and x_{12} are associated with gusts.

S. Jiffri, S. Fichera, A. Da-Ronch and J.E. Mottershead, Nonlinear control for the suppression of flutter in a nonlinear aeroelastic system, *AIAA Journal of Guidance, Control and Dynamics,* in preparation.

Pitch Linearisation

Input-output linearisation with pitch displacement chosen as the output y

$$z_1 = y = x_1 \qquad z_2 = z'_1 = y' = x'_1$$
$$z'_2 = y'' = x'_2 = f_2 \mathbf{X} + g_2 u$$

Relative degree r=2. When r is less than the number of states the system can only be partly linearised.

In matrix form,

$$\begin{cases} z_1' \\ z_2' \end{cases} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} \begin{cases} z_1 \\ z_2 \end{cases} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} v, \qquad v = f_2 \mathbf{X} + g_2 u,$$

Linear and nonlinear terms are located in $f_2(\mathbf{x})$. ν denotes the artificial input – it is the term that achieves the desired linear control objective.

Pole Placement

The artificial input may be defined as,

$$v = -k_1 z_1 - k_2 z_2,$$

Choosing $k_1 = \omega_n^2$, $k_2 = 2\zeta_\alpha \omega_n$ will assign the natural frequency ω_n

and damping ζ_{α} . Then,

$$\begin{cases} z_1' \\ z_2' \end{cases} = \begin{bmatrix} 0 & 1 \\ -k_1 & -k_2 \end{bmatrix} \begin{cases} z_1 \\ z_2 \end{cases}.$$

And the physical nonlinear input is given by,

$$u = \frac{1}{g_2} v - f_2 \mathbf{x} = \frac{1}{g_2} - k_1 z_1 - k_2 z_2 - f_2 \mathbf{x}$$

This input cancels the system dynamics and implements the linear control requirement.

Internal Dynamics

A linear coordinate transformation is carried out to obtain the system in *Normal* Form – such that the input u does not appear explicitly,

$$z = Tx,$$
 $T_{j,j} = 1$ $j = 1:12,$
 $T_{4910,:} g = 0$

remaining terms equal to 0.

The zero dynamics is then obtained by setting the controlled coordinates to zero, i.e. $z_1 = z_2 = 0$,

$$z'_{3} = z_{4}, \quad z'_{4} = -\frac{g_{4}}{g_{2}}f_{2} \quad \mathbf{Z} + f_{4} \quad \mathbf{Z} \quad , \quad z'_{5} = -\varepsilon_{1}z_{5}, \quad z'_{6} = -\varepsilon_{2}z_{6},$$
$$z'_{7} = z_{3} - \varepsilon_{1}z_{7}, \quad z'_{8} = z_{3} - \varepsilon_{2}z_{8}, \quad z'_{9} = -\frac{1}{g_{2}}f_{2} \quad \mathbf{Z} - \varepsilon_{1}z_{9},$$
$$z'_{10} = -\frac{1}{g_{2}}f_{2} \quad \mathbf{Z} - \varepsilon_{2}z_{10}, \quad z'_{11} = -\varepsilon_{3}z_{11}, \quad z'_{12} = -\varepsilon_{4}z_{12},$$

Stability of the zero dynamics must be examined to ensure the stability of the nonlinear controller.

Tuned Numerical Model

Linear frequency-domain tests:



Nonlinear time-domain tests:



Embedding the Numerical Model in the Aeroelastic Control Loop



Test Results Assigned Damping at ζ_{α} =0.3, U=15m/s



Flap motion

Feedback Linearisation Aerofoil with Non-smooth Pitch Nonlinearity



Aerodynamic model with 8 states, 6 structural (pitch, plunge, flap) and 2 aerodynamic states (Edwards, 1979).

Output y: plunge displacement ξ . Input u: flap command β_{com} .

Edwards, J. W., Ashley, H., and Breakwell, J. V. "Unsteady aerodynamic modeling for arbitrary motions," *AIAA Journal*, Vol. 17, No. 4, 1979, pp. 365-374.

S. Jiffri, P. Paoletti and J.E. Mottershead, Feedback linearization in systems with nonsmooth nonlinearities, *AIAA Journal of Guidance, Control and Dynamics*, in press.

Pitch Nonlinearity

$$\boldsymbol{f}_{nl} = \begin{cases} -\lambda K_{\alpha} \alpha, & |\alpha| \leq g_{\alpha} \\ -\lambda K_{\alpha} g_{\alpha}, & \alpha > g_{\alpha} \\ \lambda K_{\alpha} g_{\alpha}, & \alpha < -g_{\alpha} \end{cases} \} \lambda \leq 1$$

 g_{α} defines the initial (lower) stiffness region on either side of $\alpha=0^{\circ}$

Inner region $|\alpha| \le g_{\alpha}$: Net stiffness= $(1-\lambda)K_{\alpha}$ Outer region $|\alpha| > g_{\alpha}$: Net stiffness = K_{α}

 $\lambda = 1$ produces freeplay. $\lambda \leq 1$ produces piecewise nonlinearity

 K_{α} : chosen as the pitch stiffness of the desired linear system.





Feedback Linearisation Non-smooth Nonlinear System Parameters

$$\dot{\mathbf{x}} = \underline{\mathbf{f}} \ \mathbf{x} + \underline{\mathbf{g}} \ \mathbf{x} \ u, \quad \text{where} \ \underline{\mathbf{f}} \ \mathbf{x} = \begin{cases} \dot{\mathbf{q}} \\ \Psi \mathbf{q} + \Phi \dot{\mathbf{q}} + \Lambda \mathbf{q}_a + \Omega \mathbf{f}_{nl} \\ \mathbf{E}_1 \mathbf{q} + \mathbf{E}_2 \dot{\mathbf{q}} + \mathbf{F}_p \mathbf{q}_a \end{cases}, \quad \underline{\mathbf{g}} \ \mathbf{x} = \begin{bmatrix} \mathbf{0} \\ \Pi \\ \mathbf{0} \end{bmatrix}$$



No requirement to differentiate the non-smooth nonlinearity.

The transformation matrix **T** is invertible.

Input *u* eliminated in the internal dynamics

The usual smoothness requirement on the nonlinearity can be removed.

The zero dynamics are found to take the form:

found when:

$$\dot{\mathbf{z}} = \hat{\mathbf{A}} \mathbf{z} + \hat{\mathbf{b}} \psi \quad z_{3} \qquad \psi \quad z_{3} = \begin{cases} z_{3}, & |z_{3}| \le g_{\alpha} \\ g_{\alpha}, & z_{3} > g_{\alpha} \\ -g_{\alpha}, & z_{3} > g_{\alpha} \\ -g_{\alpha}, & z_{3} < -g_{\alpha} \end{cases}$$
The equilibrium points are found when:
$$\begin{cases} \hat{\mathbf{A}} + \hat{\mathbf{b}} \mathbf{e}_{1}^{T} \quad \mathbf{z}_{eq} = \mathbf{0}; \quad \mathbf{e}_{1}^{T} \mathbf{z} = z_{3} \\ \hat{\mathbf{A}} \mathbf{z}_{eq} + \hat{\mathbf{b}} g_{a} = \mathbf{0} \\ \hat{\mathbf{A}} \mathbf{z}_{eq} - \hat{\mathbf{b}} g_{a} = \mathbf{0} \end{cases}$$

Assuming non-singular $\widehat{\mathbf{A}}$ and $\widehat{\mathbf{A}} + \widehat{\mathbf{b}} \mathbf{e}_1^T$ and solving above for each of the 3 conditions,

$$\mathbf{z}_{eq} = \begin{cases} \mathbf{0}, & |z_3| \le g_{\alpha} \\ -\widehat{\mathbf{A}}^{-1}\widehat{\mathbf{b}}g_{\alpha}, & z_3 > g_{\alpha} \\ \widehat{\mathbf{A}}^{-1}\widehat{\mathbf{b}}g_{\alpha}, & z_3 < -g_{\alpha} \end{cases}$$

Stability of the Plunge Zero Dynamics

There are 2 non-zero equilibria.

Lyapunov function based around the particular equilibrium point:

$$\hat{\mathbf{z}} = \mathbf{z} - \mathbf{z}_{eq}, \qquad V = \frac{1}{2} \hat{\mathbf{z}}^T \mathbf{P} \hat{\mathbf{z}}, \qquad \mathbf{P} > 0, \qquad \mathbf{P} = \mathbf{P}^T$$

The derivative of *V* for each of the three regions may be expressed as (Shevitz and Paden, 1994):

$$\dot{\tilde{V}} = \begin{cases} \hat{\mathbf{z}}^T \mathbf{P} \ \hat{\mathbf{A}} + \hat{\mathbf{b}} \mathbf{e}_1^T \ \hat{\mathbf{z}} + \mathbf{z}_{eq} \ , & |z_3| \le g_\alpha \\ \hat{\mathbf{z}}^T \mathbf{P} \ \hat{\mathbf{A}} \hat{\mathbf{z}} + \hat{\mathbf{A}} \mathbf{z}_{eq} + \hat{\mathbf{b}} g_\alpha \ , & z_3 > g_\alpha \quad \text{for} \\ \hat{\mathbf{z}}^T \mathbf{P} \ \hat{\mathbf{A}} \hat{\mathbf{z}} + \hat{\mathbf{A}} \mathbf{z}_{eq} - \hat{\mathbf{b}} g_\alpha \ , & z_3 < -g_\alpha \end{cases} \quad \text{for} \quad \begin{cases} \mathbf{z}_{eq} = \mathbf{0} \\ \mathbf{z}_{eq} = -\hat{\mathbf{A}}^{-1} \hat{\mathbf{b}} g_\alpha \\ \mathbf{z}_{eq} = -\hat{\mathbf{A}}^{-1} \hat{\mathbf{b}} g_\alpha \end{cases}$$

These derivatives must be strictly negative for asymptotic stability. 3 z_3 stability regions, for each of the 3 equilibria \rightarrow 9 equations to be investigated.

Shevitz, D., and Paden, B. "Lyapunov stability theory of nonsmooth systems," *IEEE Transactions on Automatic Control*, Vol. 39, No. 9, 1994, pp. 1910-1914.

S. Jiffri, P. Paoletti and J.E. Mottershead, Feedback linearization in systems with nonsmooth nonlinearities, *AIAA Journal of Guidance, Control and Dynamics*, in press.

Uncontrolled Nonlinear Response



(b) Piecewise Linear Stiffness

Zero-dynamics: Freeplay



Superimposed simulated time-domain responses with randomly generated initial conditions

Zero-dynamics: Piecewise Linear Stiffness



Superimposed simulated time-domain responses with randomly generated initial conditions

Time-domain Closed-loop Response with Feedback Linearisation

Reduced air speed $U^*=2.0$ Assigned frequency and damping $\omega_{n_c} = 1$ Hz, $\zeta_{\varepsilon} = 0.1$



(b) Piecewise Linear Stiffness

Commanded and Actual Flap Angles

Reduced air speed $U^*=2.0$



(b) Piecewise Linear Stiffness

High Bandwidth Morphing Actuator (HBMA) & Modular 31 Flexible Aeroelastic Wing (ModFlex)



Conclusions

- Linear flutter suppression demonstrated by active control using the method of receptances – based on data from a vibration test with no requirement to evaluate or to know the system matrices.
- Nonlinear aeroelastic system plunge nonlinearity representative of future aircraft with very flexible wings.
- Feedback linearisation demonstration of LCO suppression.
- Non-smooth nonlinearities piecewise stiffness and freeplay.
- Feedback linearisation for non-smooth nonlinearity.
- Multiple equilibria zero and nonzero.
- Numerical demonstration.
- MODFLEX wing in preparation multiple control surfaces, conventional motor-driven flaps, morphing using piezo-benders .

Selected Papers

- Y.M. Ram and J.E. Mottershead, The receptance method in active vibration control, *American Institute of Aeronautics and Astronautics Journal*, 45(3), 2007, 562-567.
- M. Ghandchi Tehrani, R.N.R. Elliott and J.E. Mottershead, Partial pole placement in structures by the method of receptances: theory and experiments, *Journal of Sound and Vibration*, 329(24), 2010, 5017-5035.
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